1 1

RADIANT PANELS

Radiant panels now handle the majority of sensible loads in the building. As the name suggests, the panels use radiant heat transfer to condition the building. Water flows from the central plant to the radiant panel. Heat transfer occurs from copper piping inside of the panel, into the space as seen in the figure below.

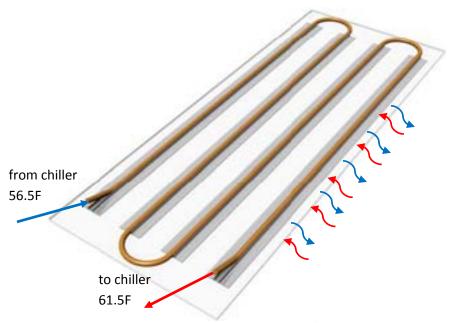


Figure 11.1. Radiant Panel shown in cooling mode^g

11.1 COOLING PANEL DESIGN

SIZING

The following table shows calculations for a few sample spaces for sizing the radiant panels for cooling. Panel ΔT is 5F, with a 56.5F inlet temperature. Total load is estimated from equation 13.1, with CFM equal to supply air at the Fan Terminal Unit for each zone and supply air as 55F. This number is conservative, since designers estimated supply air before loads were known.

Table 11.1.1. Cooling Panel Sizing

			Vent	Air Sensible	Sensible	Area	Percent
Description	Area	Load	Air	Capacity	Load	Panel ¹	Ceiling
	SF	BTU/hr	CFM	BTU/Hr	BTU/Hr	SF	
Open Office ²	593	18878	81.5	2817	16062	422	71.3%
Open Office ²	608	23803	83.6	2889	20914	550	90.5%
Open Office ²	400	13953	55.0	1901	12053	317	79.3%
Open Office ³	1878	19288	258.2	8923	10365	398	21.2%

Conference ³	400	10054	196.7	6798	3257	108	27.1%
Total For							
Building	42318	762900	5110	199000	563900	17615	41.6%

¹Appendix E

With radiant ceiling panels, there is a risk of condensation forming on the coils. If the dew point of the space reaches above the temperature of the cooling coil at the inlet (coldest point), condensation will begin to form. This would eventually cause water to drip onto occupants and equipment. The room design conditions for cooling are as follows:



Figure 11.1.2. Radiant panel in an office^h

Table 11.1.2. Room Conditions

Dry Bulb	Relative Humidity	Humidity Ratio	Dew Point	Chilled Water
74F ¹	50% ¹	.0089 lb/lb	54.3F	56.5F ²

¹Table 6.2.1

The dew point of the space is below the coldest temperature of the radiant panel. As long as design conditions are maintained, there will be no condensation. For design conditions to be met, any latent load must be handled. Below is a sample latent load calculation from a space with the highest latent load - a 400 square foot conference room. As shown, the latent capacity of the supply air is plenty to handle peak latent load.

Latent Load =
$$37 \text{ BTU/hr/person*occupancy}$$
 (11.1.1)

Latent Load = 37*26.7 = 988 BTU/hr

Latent Capacity =
$$4838 * CFM * (W_{room}-W_{supply})$$
 (11.1.2)

Latent Capacity = 4838*81.5*(.0089-.0058) = 2950 BTU/hr

²Perimeter Zone

³Interior Zone

²Table 12.2.2

CIRCUITS

Radiant panel circuitry must also be designed. The following equation is useful for finding flow rate.

$$GPM = BTUh/(500*\Delta T)$$
 (11.1.3)
 $GPM_{total} = 563900/500/5 = 225.5 GPM$

One would like total pressure drop in the circuit to be in the vicinity of 5' of head. The following table shows calculations for panels and flow rate. Note that each panel is 2'x4'.

Table 11.1.3. Panel and Flow Rate calculations

	Sensible	Area		Flow
Description	Load	Panel	Panels	Rate
	BTU/Hr	SF		GPM
Open Office	16062	422	53	6.42
Open Office	20914	550	69	8.37
Open Office	12053	317	40	4.82
Open Office	10365	398	50	4.15
Conference	3257	108	14	1.30
Total	563900	17615	2202	225.5

If one assumes 8 panels per circuit we get a head of 3' per circuit.

Table 11.1.4. Circuit design

Panels/Circuit	Total Circuits	Flow Rate/Circuit	Head per panel ¹	Head per circuit	
8	275	.820	.376′	3.01'	

¹Calculation Appendix

LAYOUT

Radiant panels should be spread out evenly over each zone. In perimeter zones, panels will cover most of the space, and will be much sparser in interior zones (see Table 11.1.1). Wherever radiant panels cover more than 70% of a zone, full ceiling systems will be usedⁱ. This is primarily at perimeter zones, and will also help coordinate with heating panels (see Section 11.2)

11.2 HEATING PANEL DESIGN

SIZING

There are a few key differences when sizing heating panels. In the McKinstry Oregon Headquarters, all heating load is through the building envelope, and is equal across the entire

perimeter. The eQUEST model from Technical Report 2 (Wyczalkowski) gives total heating load per floor. Panel ΔT is 40F, with an intlet of 100F. This is a large number however is acceptable providing expansion tanks are adequately sized.

Table 11.2.1. Heating Panel Sizing

		Total				Panel		Total
Fl	oor	Load	Perimeter	Load/ft	# Tubes ¹	Width ¹	Area	Flow ²
		BTU/hr	Ft	BTU/hr/ft	passes	ft	SF	GPM
	1 st	440000	625	704	10	5	3125	22
	2 nd	446000	625	714	10	5	3125	22.3

See Appendix F

Heating load is slightly higher on the second floor. This is due to roof exposure, which is greater than heat loss through slab on grade at the first floor level.

CIRCUITS

Using equation 11.1.3 we can find total flow rate for each floor. For this calculation we will assume a circuit flow of 1 GPM, and then check if head loss is appropriate.

Table 11.2.2. Heating Panel Circuit Head Calculation

_	Total	Total	Circuit		circuit	Tube	Head/	Circuit
Floor	Load	Flow	GPM	#circuits	length	length ¹	100ft ²	Head
	BTU/hr	GPM	GPM		ft	ft	ft H ₂ 0	ft H₂O
1st	440000	22	1	22	28.41	284.1	2	5.61
2nd	446000	22.3	1	22.3	28.03	280.3	2	5.68

¹Length = Perimeter/Circuits; 10 tubes/circuit

LAYOUT

Since all heat loads are on the perimeter, panel layout should be around the perimeter of the building, and work inside. The perimeter also needs to be cooled, so these panels would over lap. Here, a 4 pipe system will be used, so the panels can provide both heating and cooling.

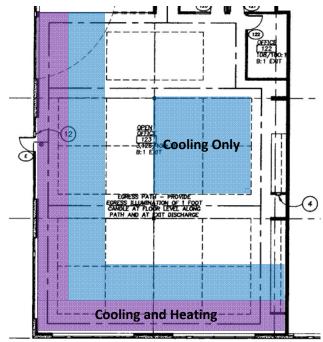


Figure 11.2.1. Radiant Ceiling Panel Layout

² Equation 11.1.3.

²See Appendix F